

# Why Did the World Trade Center Towers Collapse?

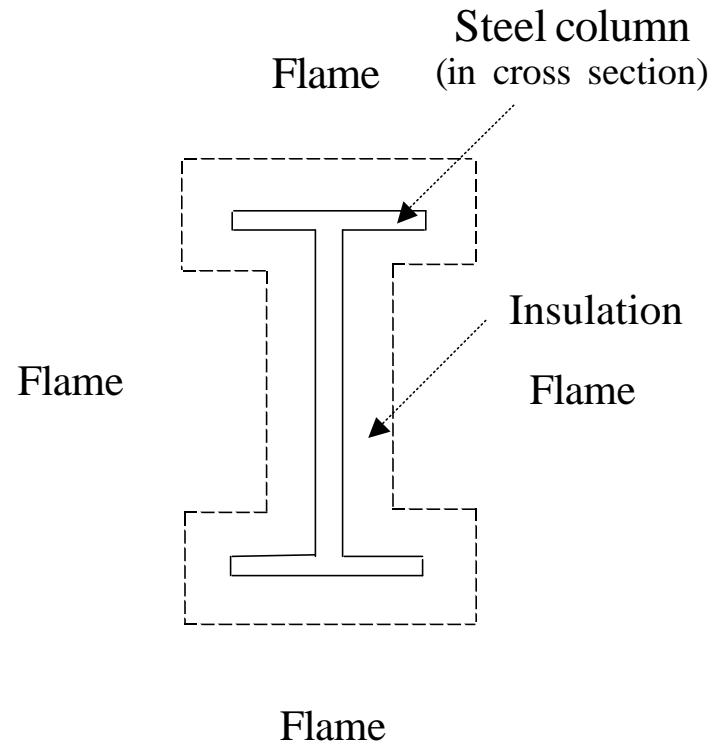
## A Numerical Heat Transfer Analysis

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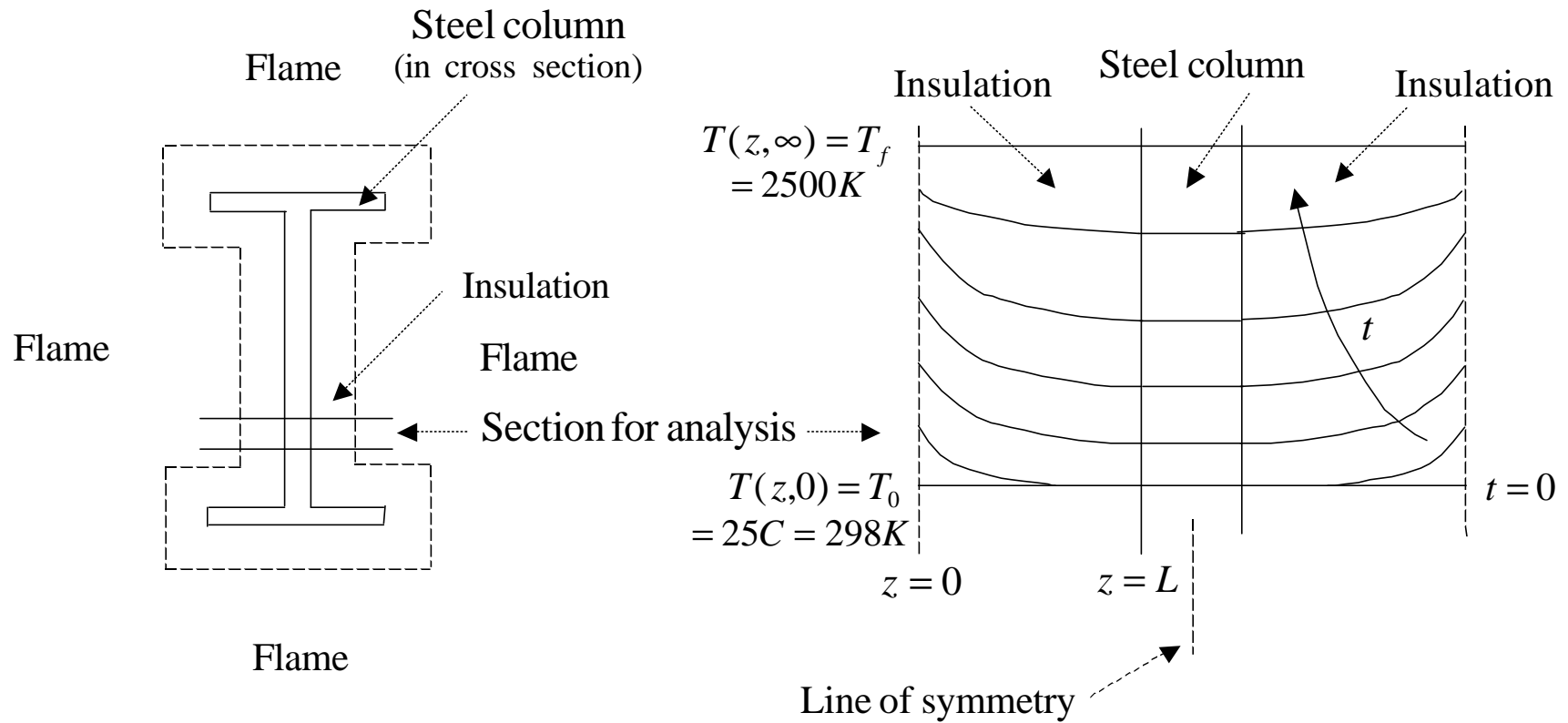
<http://www.lehigh.edu/~wes1/wes1.html>

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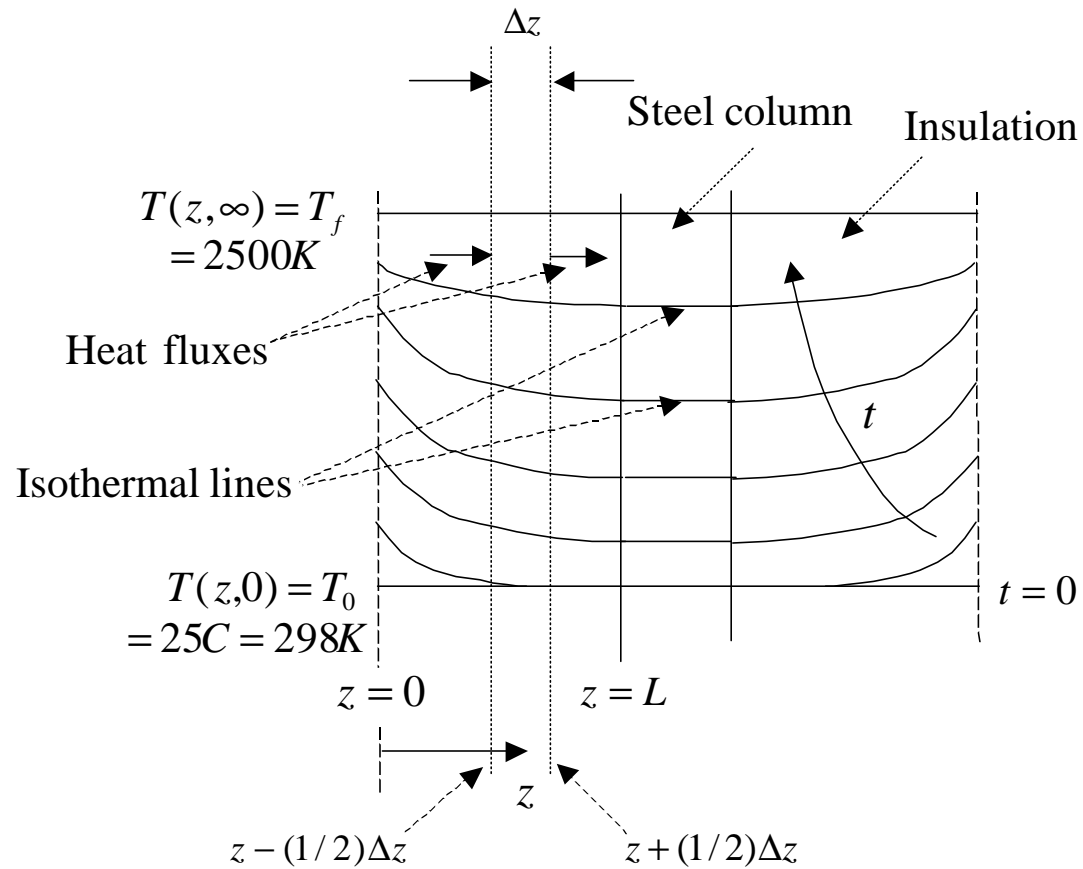
# Radiative Heating of Insulated Steel



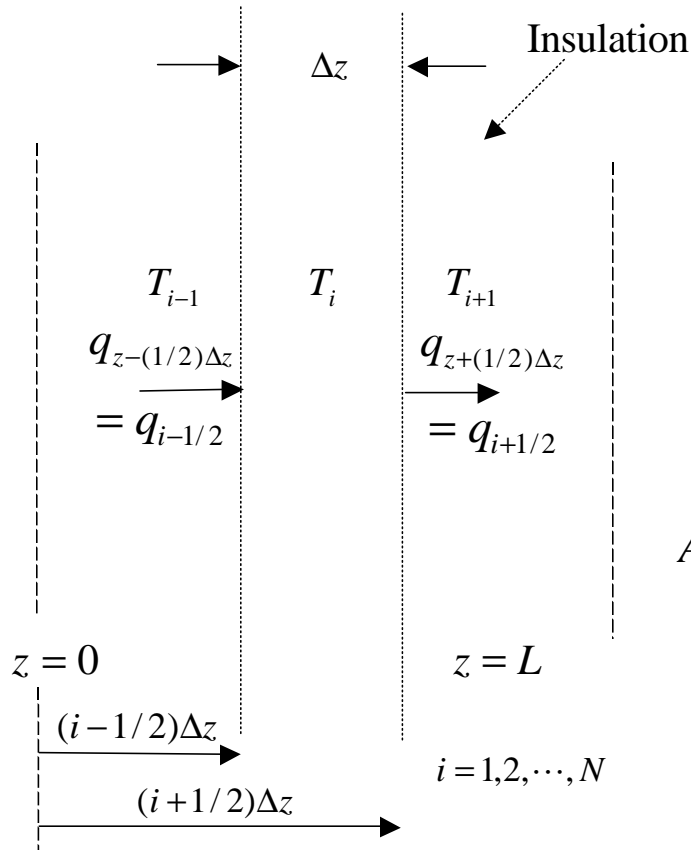
# Radiative Heating of Insulated Steel



# Energy Conservation Analysis by Finite Differences/Volumes



# Energy Conservation Analysis by Finite Differences/Volumes



By energy conservation  
on a finite volume

accumulation = input - output

$$A\Delta z r C_p \frac{dT_i}{dt} = A(q_{i-1/2} - q_{i+1/2})$$

By a FD approximation to Fourier's first law

$$q_{i+1/2} = -k \frac{(T_{i+1} - T_i)}{\Delta z} \quad q_{i-1/2} = -k \frac{(T_i - T_{i-1})}{\Delta z}$$

$$A\Delta z r C_p \frac{dT_i}{dt} = A \left[ k \left( \frac{T_{i-1} - T_i}{\Delta z} \right) - k \left( \frac{T_i - T_{i+1}}{\Delta z} \right) \right]$$

$$\frac{dT_i}{dt} = \frac{k}{r C_p} \left[ \left( \frac{T_{i-1} - T_i}{\Delta z} \right) - \left( \frac{T_i - T_{i+1}}{\Delta z} \right) \right]$$

$$\frac{dT_i}{dt} = a \left( \frac{T_{i+1} - 2T_i + T_{i-1}}{\Delta z^2} \right)$$

For  $\Delta z \rightarrow 0$ , Fourier's second law :  $\frac{\partial T}{\partial t} = a \frac{\partial^2 T}{\partial z^2}$   
One IC, 2 BCs

## Summary of Mathematical Model for Radiation Heating

$$-k \frac{\partial u(0,t)}{\partial z} = \mathbf{s} (a u_a^4 - e u^4(0,t))$$

Nonlinear radiation BC

$$z = 0$$

$$\frac{\partial u}{\partial t} = \mathbf{a} \frac{\partial^2 u}{\partial z^2}$$

$$z = L$$

$$L_s \mathbf{r}_s C_{ps} \frac{du_s}{dt} = -k \frac{\partial u(L,t)}{\partial z}$$

ODE/Neumann BC

$u_a$  ambient temperature (K)

$\mathbf{s}$  Stefan - Boltzmann constant ( $= 5.67 \times 10^{-8} \frac{\text{J}}{\text{s} \cdot \text{m}^2 \cdot \text{K}^4}$ )

$a$  absorptivity

$e$  emissivity

$k$  insulation thermal conductivity ( $\frac{\text{J} \cdot \text{m}}{\text{s} \cdot \text{m}^2 \cdot \text{K}}$ )

$\mathbf{a}$  insulation thermal diffusivity ( $\frac{\text{cm}^2}{\text{s}}$ )

$L$  insulation thickness (m)

$L_s$  steel half thickness (m)

$\mathbf{r}_s$  steel density ( $\frac{\text{kg}}{\text{m}^3}$ )

$C_{ps}$  steel specific heat ( $\frac{\text{J}}{\text{kg} \cdot \text{K}}$ )

## Mathematical Model for the Heating of Insulated Steel

The basic energy balance is Fourier's second law

$$\frac{\partial u}{\partial t} = \alpha \frac{\partial^2 u}{\partial z^2} \quad (1)$$

The initial and boundary conditions are:

$$u(z, 0) = u_o \quad (2)$$

$$-k \frac{\partial u(0, t)}{\partial z} = \sigma (au_a^4 - eu^4(0, t)) \quad (3)$$

$$L_s \rho_s C_{ps} \frac{du_s}{dt} = -k \frac{\partial u(L, t)}{\partial z} = 0 \quad (4)$$

where, in SI (MKS) units:

$u$	insulation temperature $K$
$t$	time $s$
$z$	position in the insulation $m$
$u_a$	ambient (flame) temperature $K$
$\alpha$	insulation thermal diffusivity $\frac{m^2}{s}$
$k$	insulation thermal conductivity $\frac{J \cdot m}{s \cdot m^2 \cdot K}$
$\sigma$	Stefan-Boltzmann constant $= 5.67 \times 10^{-8} \frac{J}{s \cdot m^2 \cdot K^4}$
$L$	insulation half thickness $m$
$a$	absorptivity
$e$	emissivity
$u_s$	steel temperature $K$
$L_s$	steel half thickness $m$
$\rho_s$	steel density $\frac{kg}{m^3}$
$C_{ps}$	steel specific heat $\frac{J}{kg \cdot K}$

To facilitate the analysis, dimensionless independent variables are defined as

$$z' = z/L, t' = t\alpha/L^2 \quad (5)(6)$$

Substitution of eqs. (5) and (6) in eq. (1)

$$\frac{\partial u}{\partial(L^2 t'/\alpha)} = \alpha \frac{\partial^2 u}{\partial(L^2 z')^2}$$

or

$$\frac{\partial u}{\partial t} = \frac{\partial^2 u}{\partial z^2} \quad (7)$$

where the prime (') has been dropped and it is understood that we are working with the dimensionless independent variables of eqs. (5) and (6).

Eqs. (2) to (4) transform to (again, with the prime finally dropped):

$$u(z, 0) = u_o \quad (8)$$

$$-\frac{k}{L} \frac{\partial u(0, t)}{\partial z} = \sigma(au_a^4 - eu^4(0, t)) \quad (9)$$

$$L_s \rho_s C_{ps} \frac{du_s}{d(L^2 t'/\alpha)} = -\frac{k}{L} \frac{\partial u(1, t)}{\partial z'} = 0$$

or

$$\frac{du_s}{dt} = -\left(\frac{kL}{\alpha}\right) \left(\frac{1}{L_s \rho_s C_{ps}}\right) \frac{\partial u(1, t)}{\partial z} = 0 \quad (10)$$

Eqs. (7) to (10) are the model equations programmed in the MOL code.

The initial parameters are



$$\begin{aligned}
k & 1.0 \frac{J \cdot m}{s \cdot m^2 \cdot K} \\
\sigma & 5.67x10^{-8} \frac{J}{s \cdot m^2 \cdot K^4} \\
\alpha & 1.0x10^{-6} m^2/s \\
L & 0.1m \\
a & 1.0 \\
e & 1.0 \\
u_a & 2500K \\
u_o & 25C \\
\rho_s & 7800.0 \frac{kg}{m^3} \\
C_{ps} & 435.0 \frac{J}{kg \cdot K} \\
L_s & 0.025m
\end{aligned}$$

For the MOL solution, we use  $n = 21, t_f$  (dimensionless) = 0.5 and output  $u(0, t), u(1/2, t), u(1, t)$  vs  $t$  (in minutes; note  $t$  (minutes) =  $t$  (dimensionless)  $L^2/\alpha/60$ ). In particular, we observe  $u(1, t)$  as a function of  $t$ .

# A Few Case Studies with the Mathematical Model

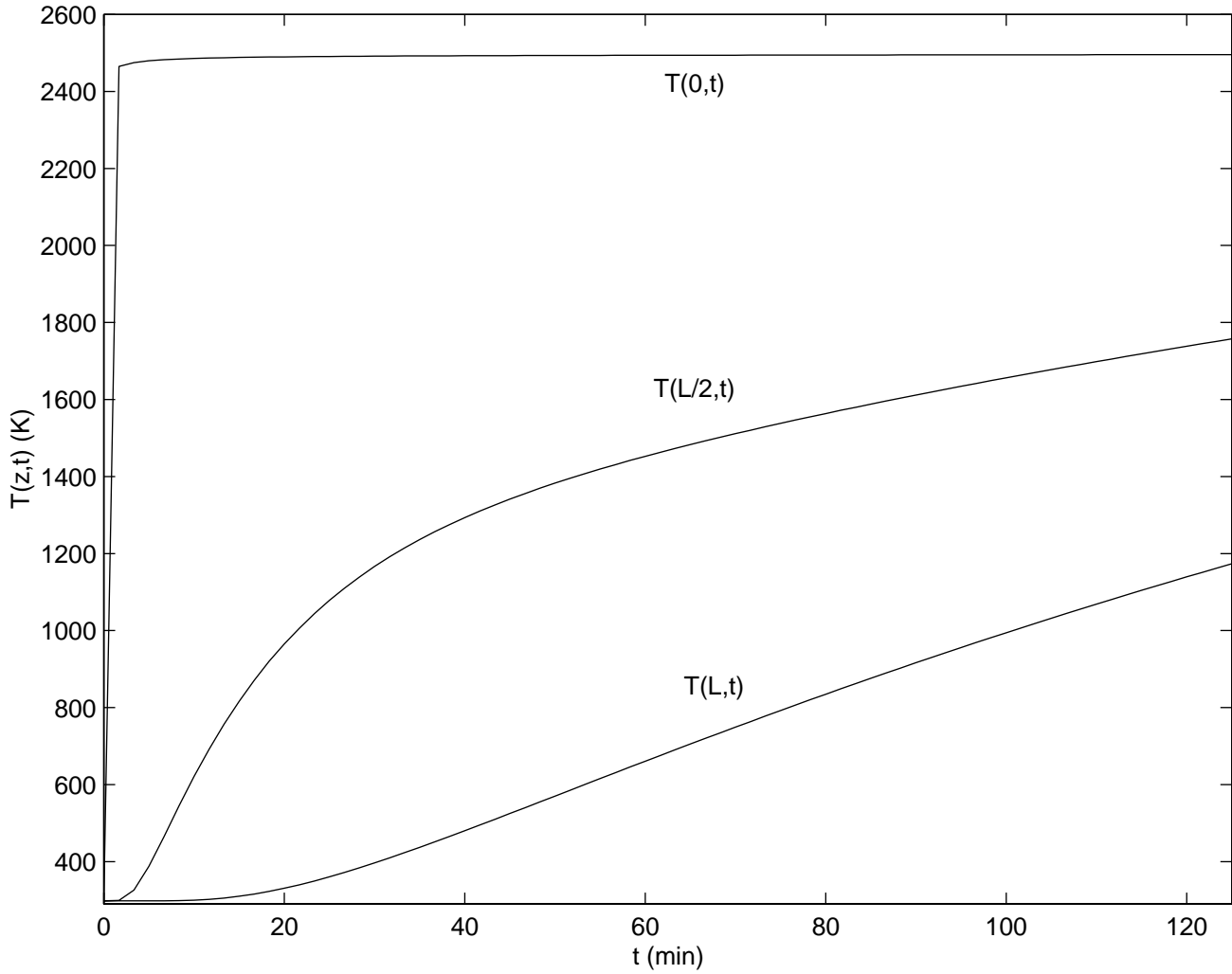
Case 1 :  $k = 1.0$ ,  $\mathbf{a} = 1.0 \times 10^{-6}$ ,  $L = 0.1$

Case 2 :  $k = 0.5$ ,  $\mathbf{a} = 0.5 \times 10^{-6}$ ,  $L = 0.1$

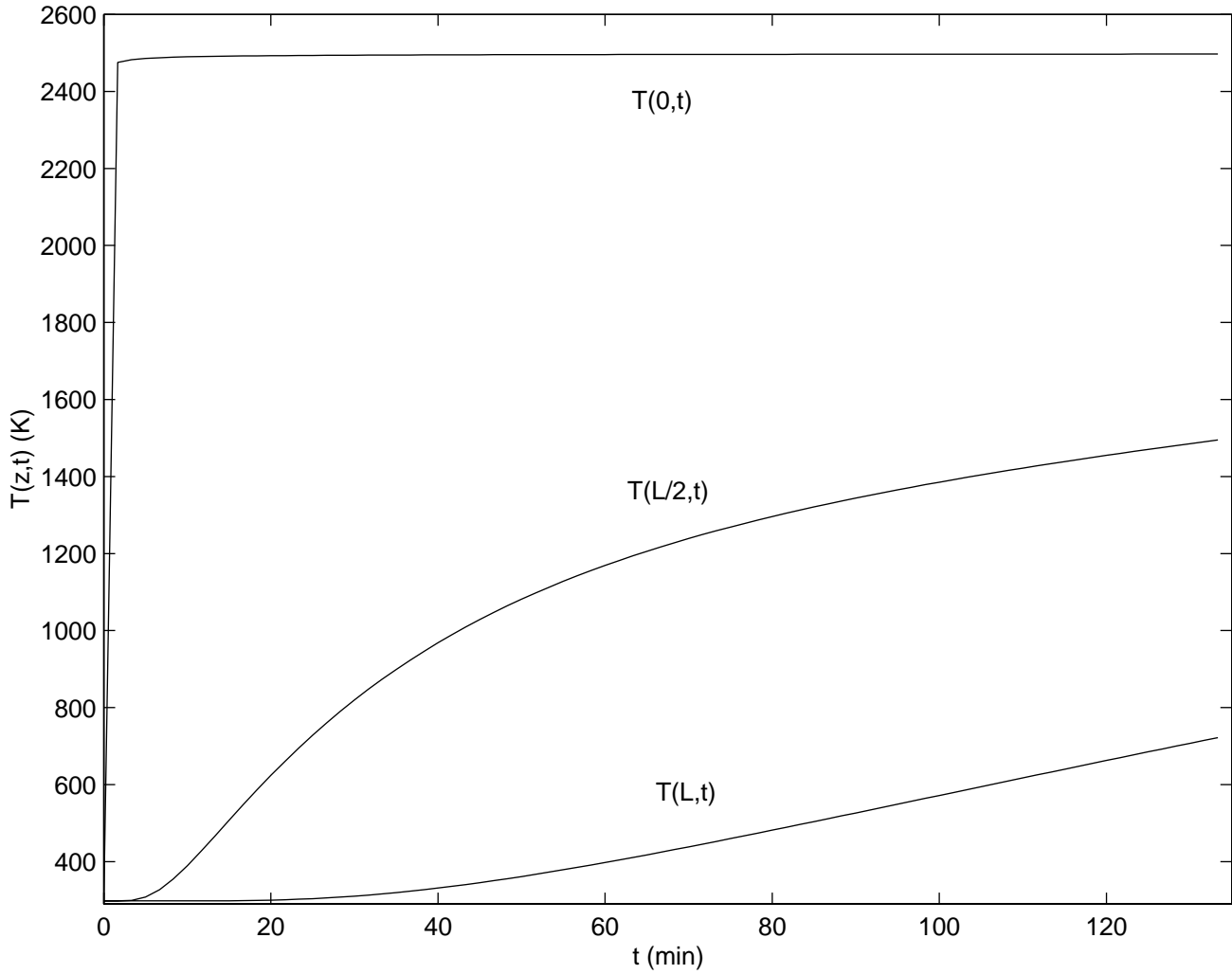
Case 3 :  $k = 1.0$ ,  $\mathbf{a} = 1.0 \times 10^{-6}$ ,  $L = 0.2$

Case 4 :  $k = 2.0$ ,  $\mathbf{a} = 2.0 \times 10^{-6}$ ,  $L = 0.1$

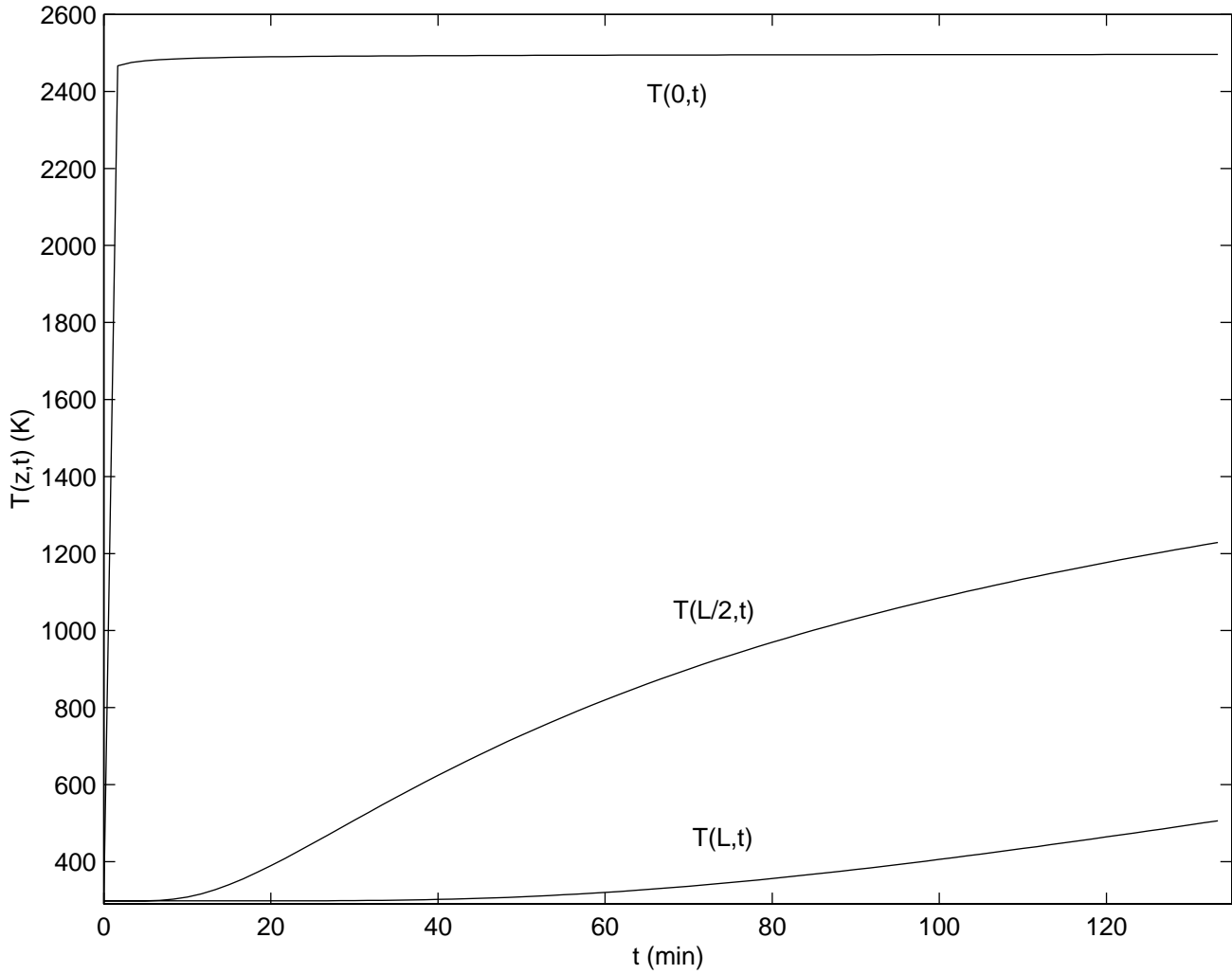
$k=1.0 \text{ J-m/s-m}^2\text{-K}$ ,  $\alpha = 1.0\text{e-}06 \text{ m}^2\text{/s}$ ,  $L = 0.1 \text{ m}$ ,  $T_f = 2500 \text{ K}$



$k=0.5 \text{ J-m/s-m}^2\text{-K}$ ,  $\alpha = 0.5\text{e-}06 \text{ m}^2\text{/s}$ ,  $L = 0.1 \text{ m}$ ,  $T_f = 2500 \text{ K}$



$k=1.0 \text{ J-m/s-m}^2\text{-K}$ ,  $\alpha = 1.0\text{e-}06 \text{ m}^2\text{/s}$ ,  $L = 0.2 \text{ m}$ ,  $T_f = 2500 \text{ K}$



$k=2.0 \text{ J-m/s-m}^2\text{-K}$ ,  $\alpha = 2.0\text{e-}06 \text{ m}^2\text{/s}$ ,  $L = 0.1 \text{ m}$ ,  $T_f = 2500 \text{ K}$

